

Microstepping Control of Ultrasonic Stepping Motors

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Abstract—The ultrasonic stepping motor (USSM) using spatially shifted standing vibrations shows the advantages of high torque, good controllability, and open-loop operation. Due to the segmentation problem of piezoelectric materials, the corresponding step size is practically limited. The purpose of this paper is to propose and implement microstepping control of this USSM. Different from the available half-step operation, the proposed control simultaneously varies both the combination of phase excitations and the magnitude of applied voltages in such a way that the desired step size can be attained. Digital implementation and experimental verification are given to validate the proposed microstepping control.

Index Terms—Motor control, piezoelectric, stepping motor, ultrasonic motor.

I. INTRODUCTION

THE traveling-wave ultrasonic motor (TWUSM) has become attractive for servomechanism, since it offers the distinct advantages of high torque density, lightweight, compact size, fast response, no electromagnetic interference (EMI), quiet operation, and power-down holding torque capability [1]–[3]. Nevertheless, the TWUSM still suffers from the drawback of closed-loop operation, thus desiring precision sensing devices [4], [5]. In order to get rid of this bulky and costly closed-loop system, the first ultrasonic stepping motor (USSM) was proposed in 1993 [6]. However, this motor generally involves many projections on the stator and many slits in the rotor, hence, suffering from the difficulty in controlling the degenerated modes [7], [8].

Recently, a viable USSM using spatially shifted standing vibrations has been proposed in which the stator does not have any projections [9]. This motor shows the definite advantages of high torque, good controllability, and open-loop operation. However, due to the segmentation problem of piezoelectric materials, which are usually made of lead zirconate titanate (PZT), the corresponding step size or the number of steps per revolution is practically limited. Very recently, an attempt has been made to improve the step size of this USSM by borrowing

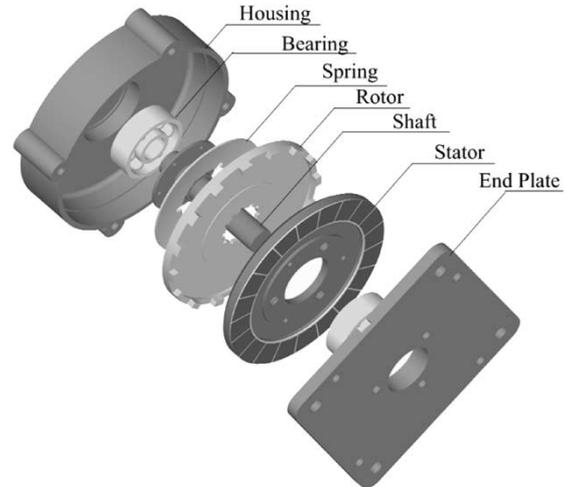


Fig. 1. USSM configuration.

the half-step operation of the conventional electromagnetic stepping motor [10].

Following the idea in [10], the purpose of this paper is to propose and implement the microstepping control of the USSM using the principle of spatially shifted standing vibrations. Different from the previous half-step operation, which successively conducts single-phase-on and two-phase-on excitation modes, the proposed microstepping control simultaneously varies both the combination of phase excitations and the magnitude of excitations in such a way that the desired step size can be attained. Different from the microstepping control of electromagnetic stepping motors in which the phase currents are modulated [11], the proposed microstepping control is to modulate the applied phase voltages.

In Section II, the operating principle of the USSM using spatially shifted standing vibrations will be briefly described. Then, the newly proposed microstepping control of this USSM will be discussed in Section III. Section IV will be devoted to present the implementation of the whole system. In Section V, experimental characteristics of the prototype will be given to verify the proposed microstepping operation. Finally, conclusions will be drawn in Section VI.

II. PRINCIPLE OF OPERATION

The configuration of the USSM using spatially shifted standing vibrations is depicted in Fig. 1. It is an 80-step ($N = 80$) USSM with the order of vibration mode $n = 8$, the number of spatial phase shifts (which is equivalent to the number of driving phases) $p = 5$, the number of electrode divisions $q = 20$, and the number of sections $m = 4$ to excite a mode

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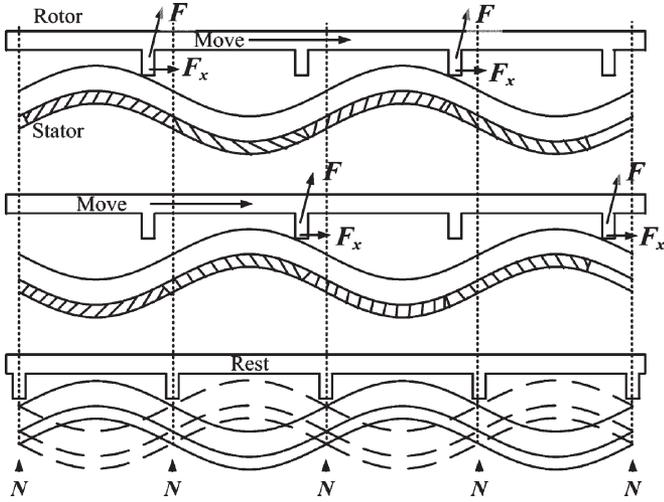


Fig. 2. Principle of stepping.

 TABLE I
 EXCITATION SEQUENCE OF FULL-STEP OPERATION

	1	2	3	4	5
A	X				
B		X			
C			X		
D				X	
E					X

each time. The rotor consists of 16 blades machined at the contact surface with the stator.

As shown in Fig. 1, the source of vibrations is the piezoelectric electrodes firmly attached beneath the stator. When these piezoelectric electrodes are excited, the stator vibrates accordingly. The spring functions to provide a constant pressure to keep the rotor onto the stator, hence enabling force transmission from the stator to the rotor. The shaft is attached with the rotor, and mounted onto the housing and end plate by two bearings.

Fig. 2 shows the principle of stepping of the USSM. When the electrodes of phase A are fed by the applied voltage, the standing-wave vibration is excited. Then, the force F is applied to the rotor by the flexural bending vibration of the stator. The corresponding horizontal component F_x pushes the rotor moving to the nodal line position of the stator. By exciting the phases in sequence, the nodal line moves along the circumference of the stator. Thus, the rotor moves with the nodal line. The sequence of excitation is shown in Table I.

The design criteria and design procedure of this USSM have been discussed in [12]. In this paper, the normal operation (namely, the small step size operation) of the USSM is considered, whereas the abnormal operation (namely, the large step size operation) is ignored. In short, the design parameters are governed by [12]

$$q = \frac{2nN}{N - 2n} \quad (1)$$

$$p = \frac{N}{2n} \quad (2)$$

$$m = \frac{q}{p} \quad (3)$$

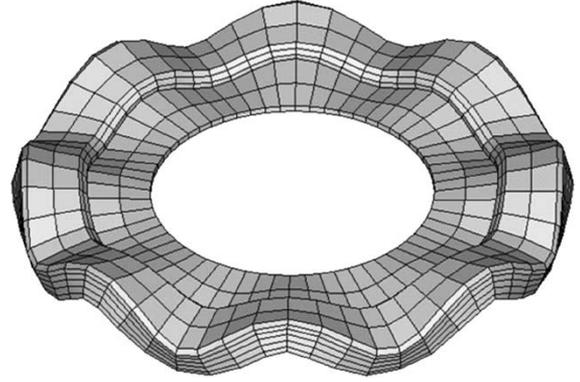


Fig. 3. FEM simulated vibration pattern.

In order to determine the driving frequency to excite the desired vibration mode, a vibration analysis is conducted. In general, a commercially available finite element method (FEM) software package can be utilized. The analysis result is shown in Fig. 3, in which the B_{08} vibration mode is adopted. Hence, the resonant frequency can be calculated as 44.9 kHz. Based on this initial estimation, the actual driving frequency is determined by trial and error, and found to be 40.5 kHz.

III. MICROSTEPPING CONTROL

In order to illustrate the proposed microstepping-control approach, three typical fractional-step operations (namely, the half-step, one-third-step, and two-third-step operations) are graphically depicted. Notice that the number of microsteps can be flexibly controlled to achieve different fractional steps. Theoretically, the size of these microsteps is unconstrained; however, its size is practically limited by the physical tolerances and manufacturing imperfections of the USSM during prototyping.

Fig. 4(a) shows the concept of half-step operation. The corresponding movement equals half of one step, hence, the term half-step operation. The corresponding excitation sequence is shown in Table II. Extending this concept, the adjustment of both the combination of phase excitations and the magnitude of applied voltages can further subdivide the original step size.

The steps from phase A to B are adopted for exemplification. When phases A and B are excited, the spatial displacement distribution of the vibration can be expressed as

$$f_a = A \sin n\theta \quad (4)$$

$$f_b = B \sin n(\theta - \theta_s) \quad (5)$$

where $\theta_s = 2\pi/N$ denotes the spatial distance between phases A and B, and A and B denote the corresponding vibration amplitudes. When phases A and B are excited simultaneously, the spatial displacement distribution can be expressed as

$$f_{ab} = f_a + f_b \quad (6)$$

Substituting (4) and (5) into (6) yields

$$f_{ab} = F_{ab} \sin n(\theta - \varphi) \quad (7)$$

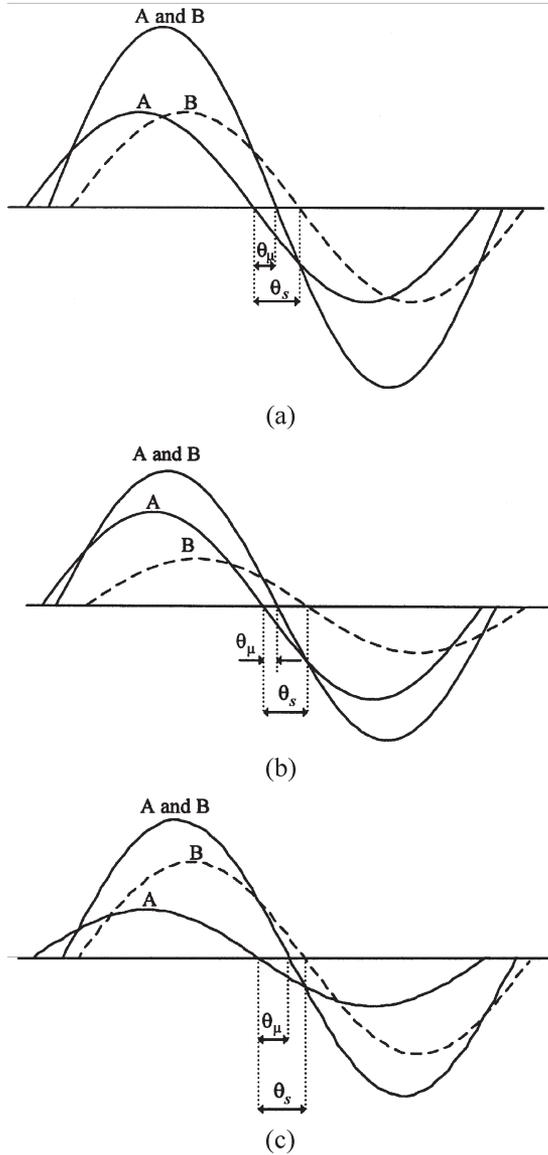


Fig. 4. Principle of microstep operations. (a) 50% step. (b) 33% step. (c) 67% step.

where the corresponding amplitude and phase difference are given by

$$F_{ab} = \sqrt{(A + B \cos n\theta_s)^2 + (B \sin n\theta_s)^2} \quad (8)$$

$$\varphi = \frac{1}{n} \arctan \left(\frac{\left(\frac{B}{A}\right) \sin n\theta_s}{1 + \left(\frac{B}{A}\right) \cos n\theta_s} \right). \quad (9)$$

Thus, when the excitation switches from phase A to the simultaneous excitation of both phases A and B, the microstep angle θ_μ is equal to the angle shift from the initial nodal line (the zero-crossing point of f_a) to the new nodal line (the zero-crossing point of f_{ab}). Hence, it yields

$$\theta_\mu = \varphi. \quad (10)$$

TABLE II
EXCITATION SEQUENCE OF 50%-STEP OPERATION

	1	2	3	4	5	6	7	8	9	10
A	X	X								X
B		X	X	X						
C				X	X	X				
D						X	X	X		
E								X	X	X

TABLE III
EXCITATION SEQUENCE OF 33%-STEP OPERATION

	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15
A	X	X	X							0	1	2	3	4	5
B		X	X	X	X	X									X
C					X	X	X	X	X						
D								X	X	X	X	X			
E											X	X	X	X	X

TABLE IV
EXCITATION SEQUENCE OF 67%-STEP OPERATION

	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15
A	X	X								0	1	2	3	4	5
B		X	X												X
C			X	X	X										
D					X	X									
E						X	X	X							
A								X	X						
B									X	X	X				
C										X	X				
D											X	X	X		
E													X	X	X

Substituting (9) into (10), θ_μ can be expressed as

$$\theta_\mu = \frac{1}{n} \arctan \left(\frac{\left(\frac{B}{A}\right) \sin n\theta_s}{1 + \left(\frac{B}{A}\right) \cos n\theta_s} \right). \quad (11)$$

After rearrangement, the voltage-amplitude ratio (B/A) can be expressed in terms of θ_μ as given by

$$\left(\frac{B}{A}\right) = \frac{\tan n\theta_\mu}{\sin n\theta_s - \tan n\theta_\mu \cos n\theta_s}. \quad (12)$$

It indicates that once the microstep size is given, there is an appropriate voltage-amplitude ratio. Fig. 4 illustrates the realization of the 50% step (half-step), 33% step (one-third step), and 67% step (two-third step) of the USSM when the values of (B/A) are chosen as 1, 0.5, and 2, respectively. The corresponding excitation sequences are tabulated in Tables II–IV, respectively.

From Tables II and III, the numbers of microsteps within a full step are 2 and 3, respectively. Since they are both integers, the excitation sequences are repeated after each cycle. On the other hand, from Table IV, the number of microsteps within a full step is 3/2, so that the excitation sequence can only be repeated after two cycles. Of course, an integer multiple of microsteps within a full step is preferred.

By using (11), when $n = 8$ and $\theta_s = 4.5^\circ$, the relationship between θ_μ and (B/A) is plotted in Fig. 5, which can provide

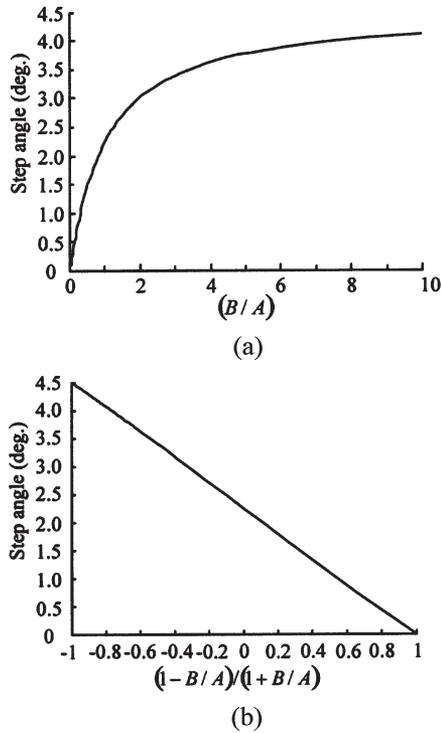


Fig. 5. Microstep angle versus voltage-amplitude ratio. (a) Nonlinear relation with (B/A) . (b) Almost linear relation with $(1 - B/A)/(1 + B/A)$.

TABLE V
PHYSICAL PARAMETERS OF PROTOTYPE

Initial number of steps	80
Initial step size	4.5°
Order of vibration mode	8
Number of driving phases	5
Number of electrodes	20
Number of blades	16

TABLE VI
GEOMETRIC PARAMETERS OF PROTOTYPE

Inner radius of stator	22.5 mm
Outer radius of stator	30.0 mm
Height of stator	3.0 mm
Height of electrodes	0.5 mm
Inner radius of rotor	26.0 mm
Outer radius of rotor	29.0 mm
Height of rotor	3.0 mm
Circumferential width of blades	2.8 mm
Radial width of blades	3.0 mm
Height of blades	1.8 mm
Radius of shaft	4.0 mm
Height of shaft	36.0 mm

TABLE VII
SPECIFICATIONS OF PROTOTYPE

Driving frequency	40.5 kHz
Applied voltage	70.7 V
Rated torque	2.8 Ncm
Maximum torque	5.6 Ncm
Maximum response time	80 ms

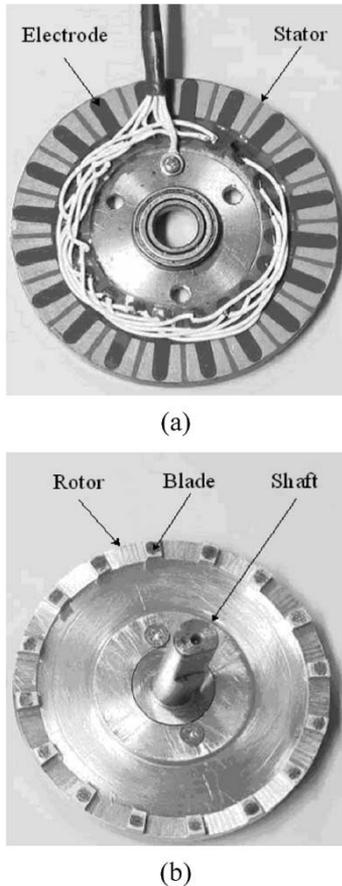


Fig. 6. USSM prototype. (a) Stator with 20 electrodes. (b) Rotor with 16 blades.

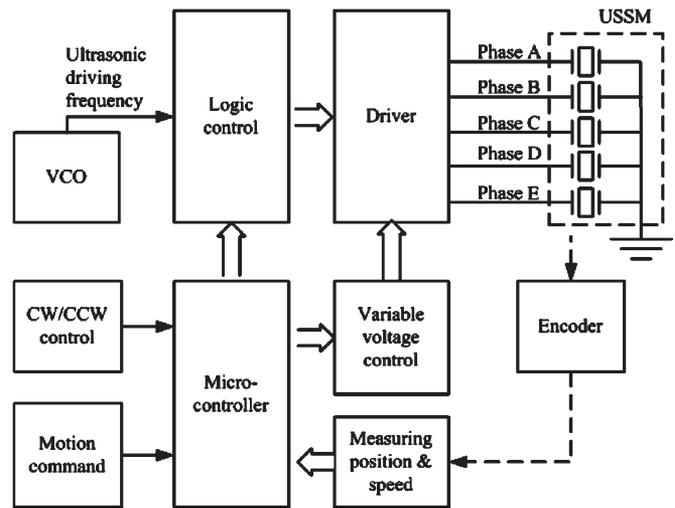


Fig. 7. Control and driving-circuit block diagram.

the designer an easy tool to select number of microsteps and the desired applied voltage amplitudes. It can be seen that the lower the (B/A) is selected, the smaller the θ_μ , and hence, resulting in the larger number of microsteps.

IV. IMPLEMENTATION

The USSM configuration shown in Fig. 1, namely, $n = 8$, $q = 20$, $p = 5$, and $m = 4$, is prototyped. The corresponding

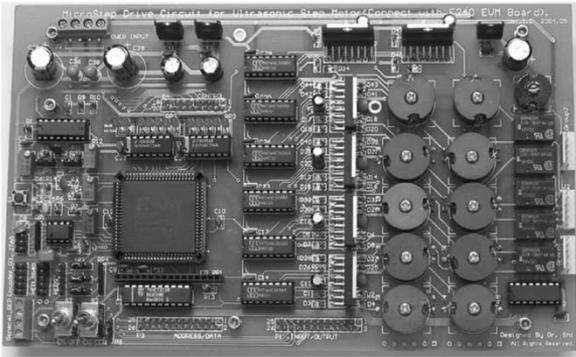


Fig. 8. Control and driving-circuit prototype.

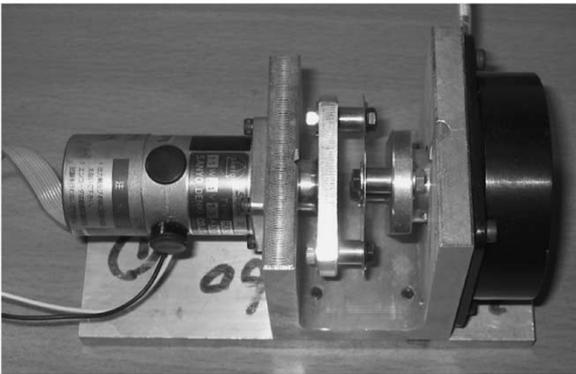


Fig. 9. Test bed of prototype.



Fig. 10. Measured vibration pattern using laser Doppler vibrometer.

structure is shown in Fig. 6. Since the elastic modulus of phosphor bronze is insensitive to the varying temperature, it is selected as the material of the elastic ring of the stator. The rotor is made of aluminum because of its lightweight and easy machining properties. The physical and geometric parameters of this prototype are tabulated in Tables V and VI, respectively. Its typical specifications are also tabulated in Table VII.

To implement the microstepping movement, the control and driving-circuit block diagram shown in Fig. 7 is used. The hardware core is the low-cost TMS320F240 microcontroller, which offers the calculation of (B/A) for a given θ_μ or the number of microsteps, the output interfaces for generating control signals, as well as the input interfaces for receiving command and measured data. The motion command can be any pattern of stepping motions with the direction governed by the clockwise/

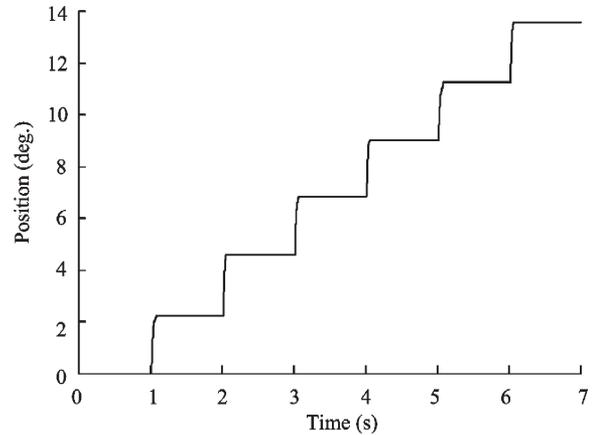


Fig. 11. Measured position response under 50%-step operation for a command of six forward steps.

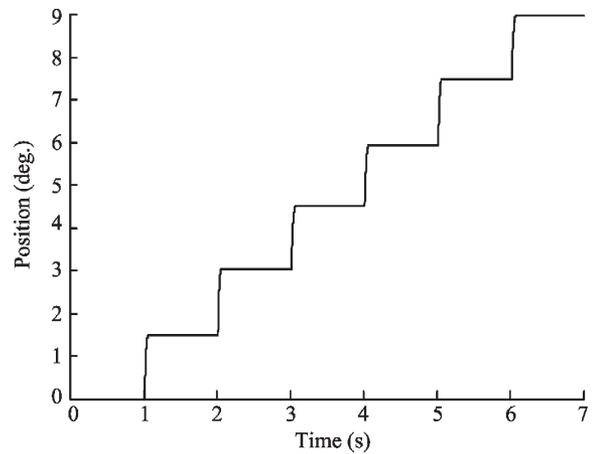


Fig. 12. Measured position response under 33%-step operation for a command of six forward steps.

counterclockwise (CW/CCW) control unit. The ultrasonic driving frequency is generated by a voltage-controlled oscillator (VCO) unit. The variable voltage control unit is to generate the applied voltages based on the calculated (B/A) . After feeding the switching control signals and driving frequency into the logic control unit, the desired logic control signals result. Then, these logic control signals and the variable voltage control signal are amplified to drive the USSM. It should be noted that the whole system is under open-loop operation. The encoder—and, hence, the measured position—and speed data are used for display only. Fig. 8 shows the prototype of this control and driving circuit.

Fig. 9 shows the test bed of the prototype for evaluation. A dc servomotor is coupled to the USSM and works as the dynamometer. The load torque of this test bed can be flexibly controlled by electrically connecting a programmable electronic load to this dc servomotor.

V. EXPERIMENTAL CHARACTERISTICS

Firstly, the vibration mode, namely B_{08} , is experimentally verified by using a laser Doppler vibrometer. Fig. 10 shows the measured spatial distribution of the vibration when phase A is

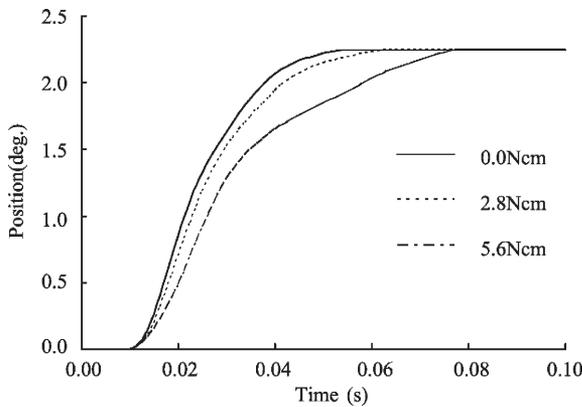


Fig. 13. Measured transient position response under 50%-step operation at different loads.

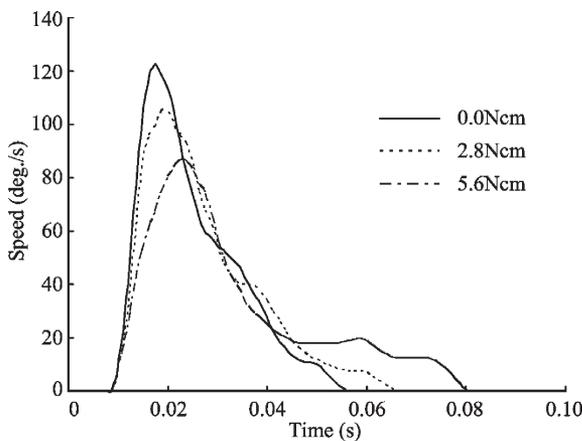


Fig. 14. Measured transient speed response under 50%-step operation at different loads.

excited, which agrees with the FEM simulated vibration pattern shown in Fig. 3.

Secondly, the dynamic characteristics of the proposed microstepping-control approach for the USSM are investigated. Two integer multiples of microsteps, namely the 50%-step and 33%-step operations, are implemented. Fig. 11 shows the position response for a command of six forward steps when microstepping control is activated for a 50%-step operation. Similarly, the position response under 33%-step operation is shown in Fig. 12. It is obvious that the measured responses well verify the accuracy of microstep angles (2.25° for 50%-step operation and 1.5° for 33%-step operation).

Finally, the transient performances of the proposed approach for the USSM are investigated. A step command is adopted for illustration. Figs. 13 and 14 show the measured transient position and speed responses at different load torques, respectively. It can be found that the response times are all less than 80 ms, confirming that the USSM can offer excellent transient performances.

VI. CONCLUSION

In this paper, a new microstepping-control approach has been proposed and implemented for the USSM using spatially shifted standing vibrations. The key is to simultaneously adjust both the combination of phase excitations and the correspond-

ing voltage-amplitude ratio in such a way that the desired step size can be attained. Digital implementation and experimental characteristics of the USSM at the 50%-step and 33%-step operations are given to verify the proposed microstepping control.

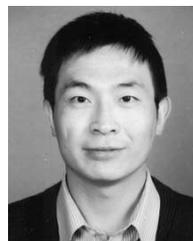
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